Optimal PI and PID Control of First-Order Plus Delay Processes and Evaluation of the Original and Improved SIMC Rules

Chriss Grimholt and Sigurd Skogestad

May 18, 2018

The first-order plus delay process model with parameters k (gain), τ (time constant) and θ (delay) is the most used representation of process dynamics. This paper has three objectives. First, we derive optimal PI- and PID-settings for this process. Optimality is here defined as the minimum Integrated Absolute Error (IAE) to disturbances for a given robustness level. The robustness level, which is here defined as the sensitivity peak (M_s) , may be regarded as a tuning parameter. Second, we compare the optimal IAE-performance with the simple SIMC-rules, where the SIMC tuning parameter τ_c is adjusted to get a given robustness. The "original" SIMC-rules give a PI-controller for a first-order with delay process, and we find that the SIMC PI-controller is close to the optimal PI-controller for most values of the process parameters (k, τ, θ) . The only exception is for delay-dominant processes where the SIMC-rule gives a pure integrating controller. The third objective of this paper is to propose and study a very simple modification to the original sime-rule, which is to add a derivative time $\tau_d = \theta/3$ (for the serial PID-form). This gives performance close to the IAE-optimal PID also for delay-dominant processes. We call this the "improved" SIMC-rule, but we put "improved" in quotes, because this controller requires more input usage, so in practice the original SIMC-rule, which gives a PI-controller, may be preferred.

1 Introduction

The PID controller is by far the most common controller in industrial practice. However, although it has only three parameters, it is not easy to tune unless

one uses a systematic approach. The first PID tuning rules were introduced by Ziegler and Nichols (1942). Although some other empirical rules were suggested, the Ziegler-Nichols (ZN) rules remained for about 50 years as the best and most commonly used rules. However, there are at least three problems with the ZN-rules:

- 1. The zN-settings are rather aggressive for most processes with oscillations and overshoots.
- 2. The zn-rule contains no adjustable tuning parameter to adjust the robustness and make it less aggressive.
- 3. For a pure time delay process, the ZN-PID settings give instability and the ZN-PI settings give very poor performance (also see discussion section).

For many years there was almost no academic interest in revisiting the PID controller to obtain better tuning rules. Dahlin (1968) considered discrete-time controllers and introduced the idea of specifying the desired closed-loop response and from this backing out the controller parameters. Typically, a first-order response is specified with closed-loop time constant τ_c (called λ by Dahlin). Importantly, τ_c (or λ) is a single tuning parameter which the engineer can use to specify how aggressive the controller should be. For first or second-order plus delay processes, the resulting controller can be approximated by a PID controller. This idea is also the basis of the internal model control (IMC) PID-controller of Rivera et al. (1986) which results in similar PID tuning rules. The IMC PI-tuning rules, also known as lambda tuning, became widely used in the pulp and paper industry around 1990 (Bialkowski, 1996).

However, the Dahlin and IMC rules set the controller integral time equal to the dominant process time constant (τ_i = τ) and this means that integral action is effectively turned off for "slow" or "integrating" processes with a large value of τ . This may be acceptable for setpoint tracking, but not for load disturbances, that is, for disturbances entering at the plant input. This led Skogestad (2003) to suggest the SIMC rule where τ_i is reduced for processes with large time constants. However, to avoid slow oscillations it should not be reduced too much, and this led to the SIMC-rule $\tau_i = \min(\tau, 4(\tau_c + \theta))$, where θ is the effective time delay of the process.

Since about 2000, partly inspired by the work of Åström (e.g., O'Dwyer (1988); Åström et al. (1992)), there has been a surge in academic papers on PID

control as can be seen by the Handbook on PID rules by O'Dwyer (2006) which lists hundreds of tuning rules.

In particular, the very simple SIMC PID tuning rules (Skogestad, 2003) have found widespread industrial acceptance. However, there has also been suggestions to improve the SIMC rules (Haugen, 2010; Lee et al., 2014). One question then naturally arises: Is there any point in searching for better PID rules for first-order plus delay processes, or are the SIMC rules good enough? To answer this question, we want in this paper to answer the following three more detailed questions: 1. What are the optimal PI and PID settings for a first-order with delay process? 2. How close are the simple SIMC rules to these optimal settings? 3. Can the SIMC rules be improved in a simple manner?

We consider the stable first-order plus time delay processes

$$G(s) = \frac{ke^{-\theta s}}{(\tau s + 1)},\tag{1}$$

where k is the process gain, τ is the process time constant, and θ is the process time delay. We mainly consider the serial (cascade) form PID controller,

$$K_{\text{PID}}(s) = \frac{k_c(\tau_i s + 1)(\tau_d s + 1)}{\tau_i s},\tag{2}$$

where k_c , τ_i and τ_d are the controller gain, integral time and derivative time. The main reason for choosing this form is that the SIMC PID-rules become simpler. For the more common parallel (ideal) PID implementation

$$K_{\text{PID}}^{\text{parallel}}(s) = k_c{'}\left(1 + \frac{1}{\tau_i{'}s} + \tau_d{'}s\right),$$
 (3)

one must compute the factor $f = 1 + \tau_d / \tau_i$, and use the following settings

$$k_c' = k_c f$$
, $\tau_i' = \tau_i f$, and $\tau_d' = \tau_d / f$. (4)

For PI-control, f=0 and the two forms are identical. In addition, a filter F is added, at least when there is derivative action, so the overall controller is

$$K(s) = K_{\text{PID}}(s) F(s). \tag{5}$$

Normally, we use is a first-order filter,

$$F = \frac{1}{\tau_f s + 1}.\tag{6}$$

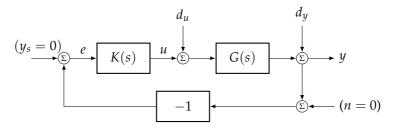


Figure 1: Block diagram of one degree-of-freedom feedback control system. We may treat a setpoint change (y_s) as a special case of an output disturbance (d_y) .

Note that τ_f is not considered a tuning parameter in this paper, but rather set at a fixed small value, depending on the case. The filter is generally needed when we have derivative action, and we may write $\tau_f = \tau_d/\alpha$ where α often is in the range from 5 to 10. For other notation, see Figure 1.

The main trade-off in controller design is between the benefits of high controller gains (performance) and the disadvantages of high controller gain (robustness and input usage) e.g., (Boyd and Barratt, 1991; Kristiansson and Lennartson, 2006). In this paper, we focus on the trade-off between IAE-performance and $M_{\rm S}$ -robustness. More pricesly, we use the integrated absolute error (IAE) for combined input and output disturbances as the performance measure and obtain optimal PI and PID settings for various robustness levels, where robustness is measured in terms of the peak sensitivity ($M_{\rm S}$) The resulting Pareto-optimal trade-off between performance and robustness is subsequently used to evaluate the SIMC PI and PID rules.

The paper is structured as follows. In Section 2 we define the measures used to quantify the performance/robustness trade-off. Based, on this optimal PI and PID controllers are presented in Section 3. In Section 4 we present the SIMC rules and propose "improved" rules, referred to as *i*SIMC and *i*SIMC-PI in this paper. In Section 5, the various SIMC and the improved rules are *i*SIMC evaluated. In Section 6, we discuss input usage and some other issues.

Preliminary versions of some of the results were presented in Grimholt and Skogestad (2012a) and Grimholt and Skogestad (2013).

2 Quantifying the optimal controller

The first authors to use the terms "optimal settings" for PID-control where Ziegler and Nichols (1942) in their classical paper. Generally, it is difficult to define "optimality" of a controller, as there are many important aspects to take into consideration, including set-point response, disturbance rejection, robustness, input usage, and noise sensitivity. Often a control loop is evaluated solely on the basis of its response to a setpoint change, but in process control, disturbance rejection is usually the major concern. Another important aspect is robustness, which often is completely omitted. Åström and Hägglund (2006) emphasise the need of including all the behaviours of the control loop.

2.1 Performance

In this paper, we quantify performance in terms of the IAE,

$$IAE = \int_0^\infty |y(t) - y_s(t)| \, \mathrm{d}t. \tag{7}$$

To balance the servo/regulatory trade-off we choose a weighted average of IAE for a step input disturbance d_u (load disturbance) and a step output disturbance d_y :

$$J(p) = 0.5 \left(\frac{IAE_{dy}(p)}{IAE_{dy}^{\circ}} + \frac{IAE_{du}(p)}{IAE_{du}^{\circ}} \right)$$
(8)

where ${\tt IAE}_{dy}^{\circ}$ and ${\tt IAE}_{du}^{\circ}$ are weighting factors, and p is the controller parameters. Note that we do not consider setpoint responses, but instead output disturbances. For the system in Figure 1, the closed-loop responses in the error $e=y_s-y$ to an output disturbance d_y and to a setpoint change y_s are identical, except for the sign. The difference is that since the setpoint is known we could further enhance the setpoint performance using a two-degrees-of freedom controller (which is not considered in this paper), whereas the unmeasured output disturbance can only be handled by the feedback controller K (which is the focus of this paper). Of course, we may consider other disturbance dynamics, but step disturbances at the plant input and output give are believed to be representative for most cases.

The two weighting factors IAE° for input and output disturbances, respectively, are selected as the optimal IAE values when using PI control (as

recommended by Boyd and Barratt (1991)). To ensure robust reference PI controllers, they are required to have $M_{\rm S}=1.59^*$, and the resulting weighting factors are given for four processes in Table 1. Note that two different reference PI controllers are used to obtain the weighting factors.

2.2 Robustness

Robustness may be defined in many ways, for example, using the classical gain and phase margins, which are related to robustness with respect to the model parameters k and θ , respectively. However, as a single robustness measure, we in this paper quantify robustness in terms of $M_{\rm ST}$, defined as the largest value of $M_{\rm S}$ and $M_{\rm T}$ (Garpinger and Hägglund, 2008),

$$M_{\rm ST} = \max\{M_{\rm S}, M_{\rm T}\}. \tag{9}$$

where M_s and M_T are the largest peaks of the sensitivity S(s) and complimentary sensitivity T(s) functions, respectively. Mathematically,

$$M_{S} = \max_{\omega} |S(j\omega)| = ||S(j\omega)||_{\infty},$$

$$M_{T} = \max_{\omega} |T(j\omega)| = ||T(j\omega)||_{\infty},$$

Table 1: Reference PI-controllers and resulting weighting factors for four processes

	Outp	ut dist	urbance	Input disturbance					
Process	k_c	$ au_i$	iae_{dy}°	k_c	$k_c $				
e^{-s}	0.20	0.32	1.61	0.20	0.32	1.61			
$e^{-s}/(s+1)$	0.55	1.14	2.07	0.52	1.05	2.02			
$e^{-s}/(8s+1)$	4.00 8.00		2.17	3.33	3.67	1.13			
e^{-s}/s	0.50	∞	2.17	0.40	5.78	15.10			

 $[\]overline{IAE}_{dy}$ and \overline{IAE}_{du} are for a unit step disturbance on output (y) and input (u), respectively.

^{*} For those that are curious about the origin of this specific value $M_{\rm S}$ = 1.59, it is the resulting $M_{\rm S}$ value for a SIMC tuned PI controller with τ_c = θ on first-order plus time delay (FOPTD) process with $\tau \leq 8\theta$.

where $\|\cdot\|_{\infty}$ is the H_{∞} norm (maximum peak as a function of frequency), and the sensitivity transfer functions are defined as

$$S(s) = 1/(1+G(s)K(s))$$
 and $T(s) = 1 - S(s)$. (10)

For most stable processes, $M_{\rm S} \geq M_{\rm T}$. In the frequency domain (Nyquist plot), $M_{\rm S}$ is the inverse of the closest distance between the critical point -1 and the loop transfer function G(s) K(s). For robustness, small $M_{\rm S}$ and $M_{\rm T}$ values are desired, and generally $M_{\rm S}$ should not exceed 2. For a given $M_{\rm S}$ we are guaranteed the following gain margin (GM) and phase margin (PM),(Rivera et al., 1986).

$$ext{GM} \geq rac{M_{ ext{S}}}{M_{ ext{S}}-1} \quad ext{and} \quad ext{PM} \geq 2 \arcsin\left(rac{1}{2M_{ ext{S}}}
ight) \geq rac{1}{M_{ ext{S}}}. \tag{11}$$

For example, $M_{\rm S}=1.6$ guarantees GM ≥ 2.67 and PM $\geq 36.4^{\circ}=0.64$ rad.

2.3 Optimal controller

For a given process and given robustness level (M^{ub}), the IAE-optimal controller is found by solving the the following optimization problem:

$$\min_{p} \quad J(p) = 0.5 \left(\frac{IAE_{dy}(p)}{IAE_{dy}^{\circ}} + \frac{IAE_{du}(p)}{IAE_{du}^{\circ}} \right)$$
 (12)

subject to:
$$M_s(p) \le M^{ub}$$
 (13)

$$M_{\scriptscriptstyle \rm T}(p) \le M^{ub} \tag{14}$$

where in this paper the parameter vector p is for a PI or PID controller. For more details on how to solve the optimization problem, see Grimholt and Skogestad (2015). The problem is solved repeatedly for different values of M^{ub} . One of the constraints in (13) or (14) will be active if there is a trade-off between robustness and performance. This is the case for values of M^{ub} less than about 2 to 3. Usually the $M_{\rm S}$ -bound is active, except for integrating processes with a small M^{ub} (less than about 1.3), where the $M_{\rm T}$ -bound is active (see Figure 4, later).

In retrospect, looking at the results of this paper, we would have obtained similar optimal PI- and PID-settings for the process (1) if we only considered input disturbances for performance and only used $M_{\rm S}$ for robustness.

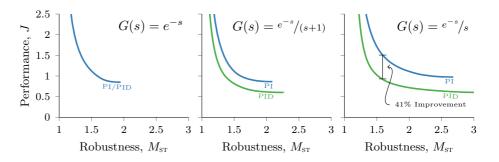


Figure 2: Pareto-optimal IAE-performance (*J*) for PI and PID control.

3 Optimal PI and PID control

3.1 Trade-off between robustness and performance and comparison of PI and PID control

In this section, we present IAE-optimal (J) settings for PI and PID controllers as a function of the robustness level ($M_{\rm ST}$). However, before presenting the optimal settings, we show in Figure 2 the Pareto optimal IAE-performance (J) as a function of robustness ($M_{\rm ST}$) for optimal PI and PID controllers for three processes. Note that the curves in Figure 2 stop when $M_{\rm ST}$ is between 2 and 3. This is because performance (J) actually gets worse and the curve for J bends upwards (Grimholt and Skogestad, 2012a) when $M_{\rm ST}$ increases beyond this value. Thus, there is no trade-off and the region with $M_{\rm ST}$ larger than about 2 should be avoided.

We see from Figure 2 that for a pure time delay process there is no advantage in adding derivative action; and it is optimal to use simple PI control. As the time constant τ increases, the benefit of using derivative action also increases. For an integrating process, derivative action improves IAE-performance by about 40%, compared to optimal PI control. This is emphasised again in Figure 3, where performance is shown as a function of the normalized time constant for robust controllers with $M_{\rm ST}=1.4$.

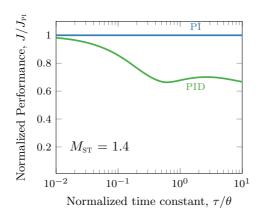


Figure 3: PID control: Normalized IAE-optimal performance for $M_{\rm ST}$ = 1.4 as a function of the normalized time constant τ/θ .

Table 2: PI control: Comparison of optimal, SIMC, and iSIMC-PI controllers with $M_{\rm ST}$ = 1.59.

	O	ptimal	PI		SII	мс						
Process	k_c	$ au_i$	J	k_c	$ au_i$	$ au_c$	J	k_c	τ_i	$ au_{\scriptscriptstyle \mathcal{C}}$	J	$M_{ m ST}$
e^{-s}	0.20	0.32	1.00	o ^a	О	1.00	1.35	0.21	0.33	0.61	1.00	1.59
$\frac{e^{-s}}{(s+1)}$	0.54	1.10	1.01	0.50	1.00	1.00	1.03	0.61	1.33	1.20	1.08	1.59
$\frac{\frac{e^{-s}}{(s+1)}}{\frac{e^{-s}}{(8s+1)}}$	3.47	4.04	1.23	4.00	8.00	1.00	1.38	4.01	8.31	1.08	1.41	1.59
$\frac{e^{-s}}{s}$	0.41	6.22	1.50	0.45	8.97	1.24	1.63	0.45	8.97	1.24	1.63	1.59

^a This is an I controller with integral gain $k_i = k_c / \tau_i = 0.5$.

3.2 Optimal PI control

The IAE-optimal PI settings are shown graphically in Figure 4 as a function of τ/θ for different robustness levels ($M_{\rm ST}$) and are also given for $M_{\rm S}=1.59$ for four processes in Table 2. For PI control we observe three main regions (Figure 4) in terms of optimal integral time τ_i :

Delay dominant: $\tau/\theta < 0.4$ $\tau_i \approx \theta/3$ Balanced: $0.4 < \tau/\theta < 4$ $\tau_i \approx \tau$ Lag dominant: t_i^a $t_i^a \approx \tau$

These regions match well the classification of first-order plus time delay processes in Garpinger et al. (2014).

In contrast with the IMC rules (Rivera et al., 1986) and the SIMC rules (Skogestad, 2003), the optimal controller does not converge to a pure integral controller ($K_{\rm I}(s)=k_i/s$, corresponding to $\tau_i\to 0$) as $\tau/\theta\to 0$ (Figure 4). Rather, for a pure time delay processes, the integral time is approximately $\theta/3$, which we will use in the proposed iSIMC-PID and iSIMC-PI rules (see below). The optimal integral time of about $\theta/3$ is almost independent of the robustness level ($M_{\rm ST}$ -values) . For balanced processes, the integral time is similar to the time constant ($\tau_i\approx \tau$, see dashed line), and also almost independent of the robustness level. This value agrees with the IMC and SIMC rules.

For lag-dominant processes (with $\tau > 4\theta$), the integral time for $M_{\rm ST} = 1.59$ approaches $\tau_i = 6.22\theta$ for $\tau/\theta = \infty$ (integrating process) (Figure 4, lower right). This is somewhat smaller than the value $\tau_i = 8\theta$ obtained from the SIMC rule. Also the normalized controller gain, $k_c k\theta/\tau$ approaches a constant value as τ goes to infinity (Figure 4, upper right). For example, for $M_{\rm ST} = 1.59$, the optimal value is $k_c k\theta/\tau = 0.414$ for $\tau/\theta = 50$, and 0.409 for $\tau/\theta = \infty$ (integrating process). This is close to the value $k_c k\theta/\tau = 0.5$ obtained with the SIMC-rule with $M_{\rm ST} = 1.59$.

3.3 Optimal PID control

The IAE-optimal PID settings are shown graphically in Figure 5 and are also given for $M_{\rm S}=1.59$ for four processes in Table 3. The optimal PID settings can be divided into the same regions as for PI control. Note that for a pure time delay process, it is optimal with PI control, that is, it is optimal to have $\tau_d=0$ and $\tau_i=\theta/3$. Actually, if we allow for having the derivative time larger than the integral time, then we could interpret it differently, and say that for a pure time delay process, the optimal controller is a integral-derivative (ID)-controller with $\tau_i=0$ and $\tau_d=\theta/3$. We will see that this latter

^a Where *k* depends on $M_{\rm ST}$ ($k \approx 6$ for $M_{\rm ST} = 1.59$).

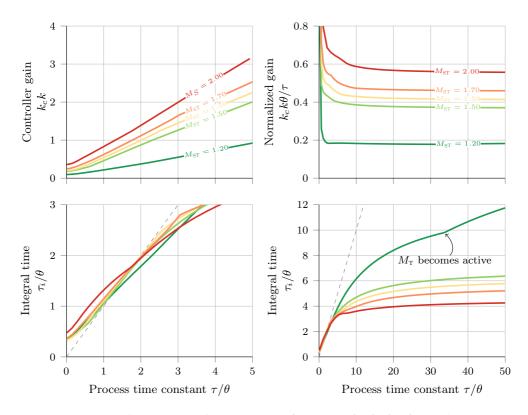


Figure 4: PI control: IAE-optimal settings as a function of τ/θ for five values of $M_{\rm ST}$.

interpretation is consistent with the proposed improved SIMC PID-rule, whereas the first interpretation is consistent with the improved SIMC PI-rule.

For PID-control, the balanced region $(0.2 < \tau/\theta 4)$ can be divided in two. In the lower part $(\tau/\theta < 1.25)$, the optimal derivative and integral time are the same, $\tau_i = \tau_d$, and increase with τ/θ . In the upper part, τ_i increases with τ/θ , whereas τ_d remains approximately constant. Note that the region with $\tau_i = \tau_d$ agrees with the recommendation of Ziegler and Nichols $(1942)^*$. However, we

^{*} Ziegler and Nichols (1942) recommend the integral time to be 4 times the derivative time for the parallel (ideal) PID controller, which for the serial (cascade) PID form corresponds to $\tau_i = \tau_d$, see (4).

Table 3: PID control: Comparison of optimal and SIMC controllers with $M_{\rm ST}$ = 1.59.

		Optin	nal PID				isimc				
Process	k_c	$ au_i$	$ au_d$	J	k_c	$ au_i$	$ au_d$	$ au_c$	J	$M_{ m ST}$	Performance (<i>J</i>) loss
e^{-s}	0.20	0.32	О	1.00	0 ^a	О	0.33	0.61	1.00	1.59	0%
$\frac{\frac{e^{-s}}{(s+1)}}{\frac{e^{-s}}{e^{-s}}}$	0.42	0.61	0.61	0.74	0.62	1.00	0.33	0.61	0.79	1.59	6%
$\frac{e^{-s}}{(8s+1)}$	4.34	2.63	0.49	0.81	4.92	6.50	0.33	0.63	1.00	1.59	23%
$\frac{e^{-s}}{s}$	0.53	3.18	0.51	0.89	0.59	6.81	0.33	0.70	1.09	1.59	22%

^a This is an ID controller with integral gain $k_i = k_c/\tau_i = 0.62$. The ID controller can be rewritten as a PI controller ($\tau_d = 0$) with $k_c = 0.62 \times 0.33 = 0.21$ and $\tau_i = 0.33$.

see from Figure 5 that $\tau_i = \tau_d$ is optimal only for a fairly small range of first-order plus time delay processes with τ/θ between about 0.2 and 1.25.

From Figure 5 we see that the integral time (τ_i) is smaller than the process time constant (τ) for all processes with $\tau/\theta > 4$, whereas we found that $\tau_i \approx \tau$ was optimal in the balanced region for PI control. For given values of $M_{\rm ST}$, the optimal PID controller gain is slightly larger than the optimal PI controller gain, and the integral action is also larger (with a lower value of τ_i).

For lag-dominant processes ($\tau/\theta > 4$), the normalized controller gain $k_c k \theta/\tau$ approaches a constant value as $\tau \to \infty$. For example, for $M_{\rm ST} = 1.59$ we have $k_c k \theta/\tau \to 0.54$. The same can be observed for the integral and derivative times which for $M_{\rm ST} = 1.59$ approach $\tau_i/\theta = 3.24$ and $\tau_d/\theta = 0.48$, respectively (Figure 5, bottom right). For increasing $M_{\rm ST}$ values (less robustness), the optimal controller gain increases and the optimal integral time decreases. Interestingly, for all lag-dominant processes the optimal derivative time is $\tau_d \approx 0.47\theta$ almost independent of the $M_{\rm ST}$ -value.

3.4 Parallel vs. serial PID controller

The above optimization was for the serial PID controller in (2). A more general PID controller is the parallel, or ideal, PID controller in (3), which allows for complex zeroes. The parallel PID controller is better only for processes with τ/θ between 0.4 and 1.2, which is the region where $\tau_i = \tau_d$ (two identical real zeros) for the serial PID controller. Furthermore, the improvement with the parallel (ideal) PID form is very minor as illustrated in

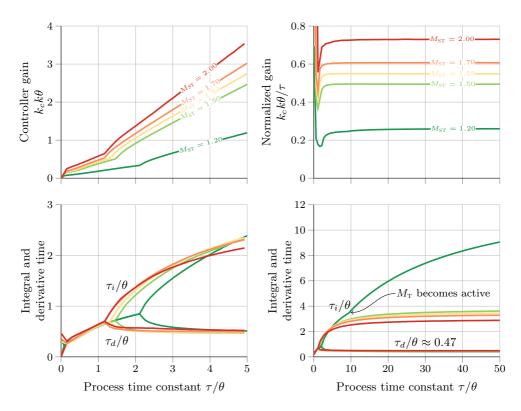


Figure 5: PID control: optimal settings as a function of τ/θ for five values of $M_{\rm ST}$.

Figure 6, which compares the IAE performance for a "balanced" process with $\tau/\theta=1$. Therefore, the serial PID implementation in (2) is sufficient for first-order plus time delay processes.

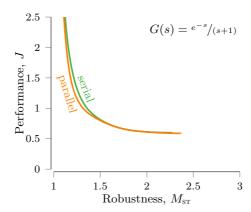


Figure 6: PID control: Comparison of IAE-optimal performance (J) for serial PID control (2) and parallel PID control (3) for a process with $\tau/\theta = 1$.

4 The original and improved SIMC rules

4.1 Original SIMC rule

We consider the first-order with delay process in (1). The original SIMC PID tunings for this process give a PI controller (Skogestad, 2003):

$$k_c = \frac{1}{k} \frac{\tau}{(\tau_c + \theta)}, \quad \tau_i = \min\{\tau, 4(\tau_c + \theta)\}.$$
 (15)

Here the closed-loop time constant τ_c is an adjustable tuning parameter which is used to get the desired trade-off between output performance, robustness and input usage. For "tight control" (good performance) with acceptable robustness (M_s about 1.6 to 1.7), Skogestad (2003) recommends selecting $\tau_c = \theta$. However, in many cases "smooth control" is desired and we should use a larger value for τ_c .

4.2 "Improved" SIMC rule with derivative action (iSIMC)

In this paper, we propose the "improved" SIMC PID-rule for a first-order with delay process. Since an important feature of the SIMC rules is simplicity, we

keep the same expressions for k_c and τ_i as in the original PI-rule in (15), but derivative action is added to improve the performance for a time delay,

*i*SIMC:
$$\tau_d = \theta/3$$
 (16)

Note that the *i*simc tunings are for the serial PID implementation in (2). For the more common parallel (ideal) PID implementation in (3), one must compute the factor $f = 1 + \tau_d/\tau_i$ and use the values in (4).

As seen from Figure 5), the value $\tau_d = \theta/3$ is close to the optimal for a pure time delay process (with $\tau = 0$). For larger values of process time constant τ , the optimal value of τ_d is closer to $\theta/2$. However, we chose to use the smaller value $\tau_d = \theta/3$ in order to reduce possible other disadvantages of adding derivative action.

If we use the same value for the tuning constant (e.g. $\tau_c = \theta$) as for the original SIMC PI controller in (15), then the addition of the derivative action in (16) mainly improves robustness (lower M_s). However, the main reason for introducing derivative action is usually to improve performance, and to achieve this one should reduce τ_c . In the original SIMC rule (Skogestad, 2003) it was recommended to select $\tau_c = \theta$ to achieve "tight control" with acceptable robustness (M_s about 1.6 to 1.7). However, as will become clearer from the results below, for the *i*SIMC PID rule we recommend reducing the value of τ_c and select $\tau_c \geq \theta/2$.

The simc pi-tunings parameters with $\tau_c = \theta$ and the *i*simc pid-tunings with $\tau_c = \theta/2$ are given for four first-order plus delay processes in Table 4.2. As seen from, *i*simc pid-improves IAE performance (*J*) by about 30% compared to the original simc pi controller, while keeping about the same robustness level ($M_{\rm ST}$ about 1.7).

Note that we have put "improved" in quotes for the iSIMC rule. Indeed, in the original SIMC paper, Skogestad (2003) considered adding the derivative time $\tau_d = \theta/2$ to counteract time delay, but concluded that it was probably not worth the increased complexity of the controller and the increased sensitivity to measurement noise and input usage. Therefore, in most practical situations in industry, the original SIMC PI-rule is most likely preferable. Nevertheless, if performance is important and τ_c is adjusted as mentioned above, then the results of this paper show (e.g. see Figure 9), that significant improvements in performance may be achieved with the iSIMC rule. Additionally, we have found that PID control with iSIMC tunings is better in almost all respects than a well-tuned Smith Predictor controller (Grimholt and Skogestad, 2018).

Table 4: SIMC PI controller and *i*SIMC PID controller with tuning constant $\tau_c = \theta$ and $\tau_c = \theta/2$, respectively.

	SIMC (PI, $\tau_c = \theta$)								i SIMC (PID, $\tau_c = \theta/2$)							
Process	k_c	$ au_i$	IAE _{dy}	IAE _{du}	J	$M_{ m ST}$		k_c	τ_i	τ_d	IAE _{dy}	IAE_{du}	J	$M_{ m ST}$		
e^{-s}	0.50 ^a	О	2.17	2.17	1.35	1.59		0.67*	О	0.33	1.50	1.50	0.93	1.66		
$\frac{e^{-s}}{(s+1)}$	0.50	1.00	2.17	2.04	1.03	1.59		0.67	1.00	0.33	1.50	1.50	0.73	1.66		
$\frac{e^{-s}}{(8s+1)}$	4.00	8.00	2.17	2.00	1.38	1.59		5.33	6.00	0.33	1.80	1.12	0.91	1.67		
$\frac{e^{-s}}{s}$	0.50	8.00	3.92	16.00	1.43	1.70		0.67	6.00	0.33	2.83	9.00	0.95	1.73		

^a Integral gain $k_i = k_c/\tau_i$.

4.3 Alternative improved SIMC rule without derivative action (*i*SIMC-PI) for delay dominant processes

Note that for a pure time delay process ($\tau=0$), the isimc Pid-controller in (15-16) is actually a id-controller, since $k_c=0$. As noted earlier this id-controller (with $\tau_d=\theta/3$ and $\tau_i=0$) may be realized instead as a pi-controller (with $\tau_i=\theta/3$ and $\tau_d=0$). This is the basis for the following "improved" simc Pi rule for a foptd process, denoted isimc-Pi (Grimholt and Skogestad, 2012b):

*i*simc-pi:
$$k_c = \frac{1}{k} \frac{\tau + \theta/3}{(\tau_c + \theta)}, \quad \tau_i = \min\{\tau + \theta/3, 4(\tau_c + \theta)\}.$$
 (17)

Note that for a pure time delay process (τ = 0), the isimc pid-controller in (15-16) and the isimc-pi pi-controller in (17) are identical. The isimc-pi tunings in (17) may give significant performance improvements benefits compared to the original simc pi-tunings for delay-dominant processes, but at the expense of larger input usage. However, for processes with $\tau > \theta/2$, approximately, we find that the benefits are marginal or even negative.

5 Evaluation of the SIMC and *i*SIMC rules

5.1 SIMC PI-rule (original)

We compare in Figure 7 the Pareto-optimal IAE performance (*J*) of the SIMC PI controller with the IAE-optimal PI controller for four different processes. The

PI settings for $M_{\rm ST}$ = 1.59 are given in Table 2. In addition, the SIMC controllers for three specific choices of the tuning parameter,

- $\tau_c = 1.5\theta$ (smoother tuning)
- $\tau_c = \theta$ (tight/redcommended tuning)
- $\tau_c = 0.5\theta$ (more aggressive tuning)

are shown by circles. For the SIMC controller (Figure 7), the trade-off curves were generated by varying the tuning parameter τ_c from a large to a small value. Except for the pure time delay process, the IAE-performance SIMC PI-controller is very close (within 10%) to the IAE-optimal PI controller for all robustness levels. In other words, by adjusting τ_c we can generate a close-to-optimal PI-optimal controller with a given desired robustness. Another important observation is that the default PID-recommendation for "tight" control, τ_c = θ (as given by middle of the three circles), in all cases is located in a desired part of the trade-off region, well before we reach the minimum. Also, the recommended choice gives a fairly constant M_s -value, in the range from 1.59 to 1.7. From this we conclude that, except for the pure time delay process, there is little room to improve on the SIMC PI rule, at least when performance and robustness are as defined above (J and M_s).

5.2 Improved SIMC PI rule (iSIMC-PI)

The main "problem" with the original SIMC rule is for pure time delay processes, where the IAE-performance (J) is about 40% higher than the optimal (Figure 7). The proposed iSIMC-PI rule in (17) rectifies this. As seen from Figure 7 (upper left), the proposed iSIMC-PI rule is almost identical to the IAE-optimal controller when τ_c is adjusted to give the same robustness ($M_{\rm ST}$). This is further illustrated by the simulation in Figure 8.

5.3 Improved SIMC PID rule (*i*SIMC)

Next, we consider PID control, that is, the addition of derivative action using $\tau_d = \theta/3$, as proposed with the *i*simc rule (16). We compare in Figure 9 the IAE performance (*J*) of the *i*simc PID controller with the IAE optimal PID controller with the same robustness ($M_{\rm ST}$) for four different processes (green curves). PID settings for $M_{\rm ST} = 1.59$ are given in Table 3. To illustrate the benefits of derivative action we also show in Figure 9 the curves with PI control (blue curves).

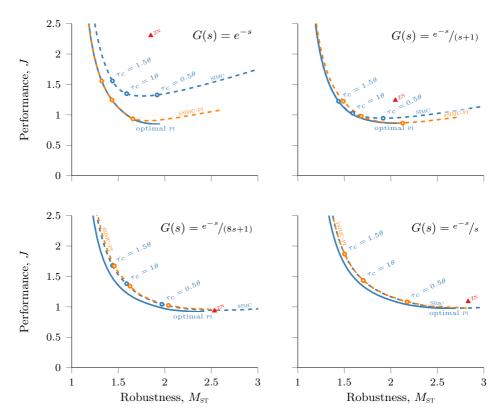


Figure 7: PI control: Pareto optimal IAE-performance (J) for optimal-PI, SIMC (15), and iSIMC-PI (17) control for four processes. The trade-off for SIMC and iSIMC-PI is generated by changing the value of the closed-loop tuning constant τ_c .

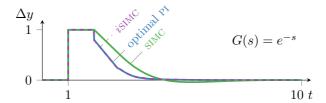


Figure 8: Time response for optimal, SIMC and iSIMC controllers ($M_{\rm ST}$ =1.59) for an input/output disturbance (at time o/1) for the pure time delay process $G(s)=e^{-s}$. For pure time delay process the input u response is equal to the output y response time shifted one time delay earlier. Also the response is the same for an input or output disturbance (but shifted). Note that for a pure time delay process there is no benefit of adding derivative action ($\tau_d=0$ is optimal).

We see from Figure 9 that the SIMC PID-controller (dashed green curve) is close to the optimal PID-controller (solid green line) for all four processes and all robustness levels. By considering the location of the middle green circles, we see that if we keep the value of τ_c unchanged at $\tau_c = \theta$, then adding derivative action mainly improves robustness. For example, for an integrating process and $\tau_c = \theta$, the value of $M_{\rm ST}$ is improved from 1.70 for PI to 1.46 for PID, but there is only a 6% improvement in performance. However, by reducing τ_c we can significantly improve performance for a given $M_{\rm ST}$ value. For the four processes, we see from Figure 9 that $\tau_c = \theta/2$ (rightmost green circles) is a good choice for the tuning constant for the *i*SIMC PID controller. Compared to SIMC PI controller tuned with $\tau_c = \theta$, this gives about 30% better IAE-performance and similar robustness ($M_{\rm ST}$ about 1.7).

We said that the SIMC PID-controller is close to the optimal PID-controller. However, we see from the two lower plots in Figure 9 that the performance loss is somewhat larger for processes with large time constants. To study this further, we compare in Figure 10 the step responses for various PI and PID controllers for an integrating process. Because of a larger integral time, the SIMC and *i*SIMC controllers settle more slowly than the optimal PI and PID controllers for both input and output disturbances. This results in a 22% higher average IAE-performance (*J*) for the *i*SIMC PID controller when compared with the optimal PID controller. However, it is usually the maximum deviation that is of main concern in industry. Because of a larger

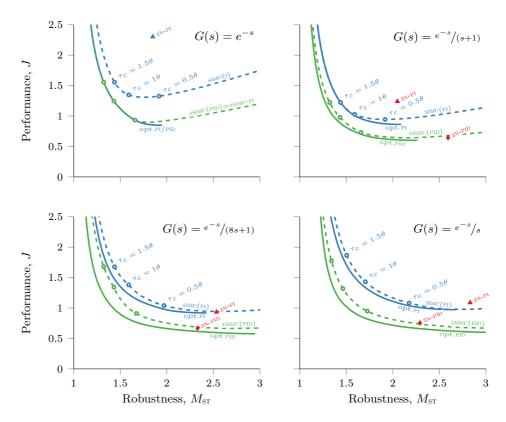


Figure 9: Pareto optimal IAE-performance (J) as a function of robustness $M_{\rm ST}$ for optimal PI and PID, and SIMC (15), iSIMC (16) and ZN for four processes. The trade-off for the various SIMC rules are generated by changing the value of the closed-loop tuning constant τ_c . PI controllers are shown with blue and PID controllers with green.

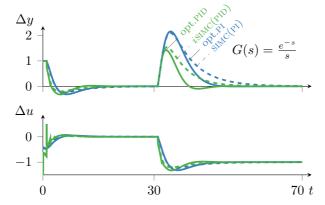


Figure 10: Time response for optimal PI and PID compared with SIMC and iSIMC controllers ($M_{\rm ST}=1.59$) for an output disturbance (at time 0) and an input disturbances (at time 20), for an integrating process $G(s)=e^{-s}/s$. To get a proper system, a first order measurement filter with $\tau_f=0.02$ was applied to the PID controllers.

controller gain, the SIMC controllers have roughly the same peak deviation as the optimal PI and PID controllers for input disturbance (see Figure 10), and a smaller overshoot for output disturbances (setpoint) than the optimal. Thus, we conclude that the performance of the SIMC controllers are better than indicated from the IAE-value (*J*), when we take into account other aspects of performance. In conclusion, also for PID control, we conclude that *i*SIMC is close to the optimal PID-controller, so the benefit of looking for even more "improved" rules for first-order plus time delay processes is limited.

6 Discussion

6.1 Input usage and filtering

Input usage is an important aspect for control, but have not been explicitly treated in this work. From Figure 1 we have

$$u = -T d_u - KS \left(d_y + n \right).$$

Thus, input usage is determined by two transfer functions: T = GK/(1 + GK) (for input disturbance) and KS = K/(1 + GK) (for output disturbance and noise). Input disturbances will not pose a new problem because T is closely related to performance and is in addition already bounded by M_T .

The important new transfer function is therefore *KS*, and by limiting its peak one can adjust input usage related to measurement noise and output disturbances (Kristiansson and Lennartson, 2002). For PI control, *KS* has a peak at intermediate frequencies which is approximately (Åström and Hägglund, 2006):

$$||KS||_{\infty}^{\text{PI}} \approx k_c M_{\text{s}}.$$
 (18)

Here, M_s is already bounded in our analysis, and is typically smaller than 1.7. Thus, we find that the controller gain k_c provides direct information about the input usage related to measurement noise and output disturbances.

For PID control, the value of |KS| is generally higher at higher frequencies, and we find the input usage is dominated by the selected measurement filter. If there is no measurement filtering, $\tau_f=0$, then the high-frequency peak goes to infinity. Therefore, for PID control it is important to filter out the high frequency noise, and the resulting peak will depend heavily on the selected filter time constant. With a first or second order measurement filter

$$F_1(s) = \frac{1}{\tau_f s + 1}$$
 or $F_2(s) = \frac{1}{(\tau_f s)^2 + \sqrt{2}\tau_f s + 1}$ (19)

the high-frequency peak can be approximated by

$$||KS||_{\infty}^{\text{PID}} \approx \alpha k_c$$
 (20)

where $\alpha = \tau_d/\tau_f$. Note that τ_f here is for the cascade PID-controller in (2) and not for the ideal form in (3). Typically, α is between 5 and 10. The ratio in input magnitude between PI and PID related to measurement noise and output disturbances can then be expressed as

$$\frac{\|KS\|_{\infty}^{\text{PID}}}{\|KS\|_{\infty}^{\text{PI}}} \approx \frac{k_c^{\text{PID}}}{k_c^{\text{PI}}} \frac{\alpha}{M_s^{\text{PI}}}.$$
 (21)

With the recommended tight tuning ($\tau_c = \theta$ for SIMC PI and $\tau_c = \theta/2$ for iSIMC PID), we get $\frac{k_c^{\text{PID}}}{k_c^{\text{PI}}} = (\theta + \theta)/(\theta/2 + \theta) = 1.33$ and the ratio in input usage can be expressed as

$$\frac{\|KS\|_{\infty}^{\text{ISIMC(PID)}}}{\|KS\|_{\infty}^{\text{SIMC(PI)}}} \approx \frac{1.33}{1.6} \alpha \approx 0.8\alpha$$
 (22)

where we have assumed $M_{\rm s}^{\rm PI}$ to be 1.6. Since α is typically larger than 5, this means that the improved IAE performance of PID control may require an input magnitude related to measurement noise and output disturbances which is at least 4 times larger than for PI control *.

Trade-off curves for *i*SIMC with different first-order measurement filters are shown in Figure 11. For the recommended PID tuning, $\tau_c = \theta/2$, performance and robustness will deteriorate with increased measurement filtering. With $\tau_c = \theta/2$ and $\alpha = 3$, the robustness is quite low, and a retuning of the controller by selecting a larger τ_c might be necessary. With α =1, we recover the original SIMC PI-controller for which we recommend $\tau_c = \theta$ to get a good trade-off between performance and robustness.

Based on Figure 11, we recommend for PID-control to choose α in the range from 5 to 10, which gives most of the benefit of the D-action. The high-frequency input usage may then increase by a factor 4 to 8 compared to PI-control. This increase in input usage may be undesirable, so for many real process applications where performance is not the key issue, the original SIMC rule, which gives a PI-controller, is the best choice.

If noise filtering is an important factor, an iterative design approach can be used in combination with SIMC to ensure that the controller has both good robustness and low noise sensitivity (Segovia et al., 2014).

6.2 Trade-off between input and output disturbance response

As already noted from Table 1 and observed from the simulations in Figure 12, the optimal controller that minimizes the average IAE performance (J) in (12), puts more emphasis on disturbance rejection at the input (IAE $_{du}$) than at the output (IAE $_{dy}$), especially for larger values of the process time constant. For example, for an integrating process we find $\widehat{IAE}_{du} = 1.02$ (close to optimal for input disturbance), whereas $\widehat{IAE}_{dy} = 1.99$ (twice the optimal). This is further illustrated in Figure 13, where we show ratio between the two terms in the IAE-performance index J (Huba, 2013), which in this paper we term *controller balance*,

controller balance =
$$\frac{IAE_{du}}{IAE_{du}^{\circ}} / \frac{IAE_{dy}}{IAE_{dv}^{\circ}},$$
 (23)

^{*} We have assumed that the selected filter does not influence controller performance and robustness in a significant way. Otherwise, we have a proportional-integral-derivative-filter (PIDF) controller where also the filter constant τ_f should be considered a degree of freedom in the optimization problem.

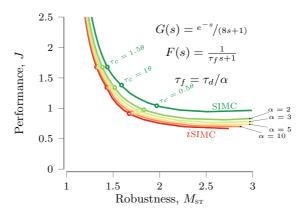


Figure 11: Performance/robustness trade-off curve for the *i*SIMC (PID) controller with measurement filtering. The SIMC (PI) controller corresponds to the case with $\alpha=1$.

as a function of τ/θ . From Figure 13, we also note that for time constants less than about 3 θ , the optimal controller has roughly equal weight on input and output (also seen in Figure 12, top). For larger time constants, the emphasis shifts towards input disturbances. Interestingly, if we use a cost function with only a small weight (1%) on input disturbances

$$J(p) = \left(0.99 \frac{\text{IAE}_{dy}(p)}{\text{IAE}_{dy}^{\circ}} + 0.01 \frac{\text{IAE}_{du}(p)}{\text{IAE}_{du}^{\circ}}\right), \tag{24}$$

we find for an integrating process the optimal settings $k_c = 0.462$ and $\tau_i = 12.2\theta$, with $\widetilde{\text{IAE}}_{du} = 1.72$ and $\widetilde{\text{IAE}}_{dy} = 1.91$. We notice that there is only a minor improvement in setpoint performance (IAE $_{dy}$ decreases about 4%), whereas disturbance rejection is much worse (IAE $_{du}$ increases about 69%). The conclusion is this that we may put emphasis mainly on input disturbances when tuning PI controllers for lag-dominated processes.

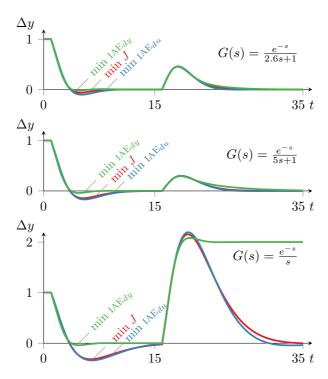


Figure 12: Comparison of step disturbance responses of IAE optimal controller which minimzes J, with the two reference controllers that consider disturbances only at the output (IAE $_{dy}^{\circ}$) and input (IAE $_{du}^{\circ}$), respectively. All controllers have $M_{\rm ST}$ = 1.59 (Table 1).

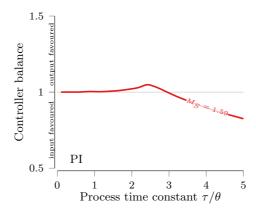


Figure 13: Controller balance between input and output disturbances, defined as $(IAE_{du}/IAE_{du}^{\circ})/(IAE_{dy}/IAE_{dy}^{\circ})$, for optimal PI control as a function of the time constant τ/θ ($M_{ST}=1.59$).

6.3 Further evaluation of SIMC PI rule for integrating processes

When comparing the optimal PI settings with the original SIMC rule for an integrating process, we find that the SIMC integral time is larger than the optimal (Figure 4, bottom). Specifically, for an integrating process with $\tau_C = \theta$ (giving $M_{\rm ST} = 1.70$), the SIMC rule gives $\tau_i/\theta = 8$, whereas the optimal performance (J) for the same robustness is with $\tau_i/\theta = 5.6$. This indicates that the SIMC rule puts more emphasis on output disturbances than input disturbances, tham for the IAE-optimal controller with equal weighting. To shift the trade-off between output (setpoint) and input disturbance, one may introduce an extra parameter in the tuning rule (Alcántara et al., 2010; Di Ruscio, 2010). Haugen (2010) suggested to introduce an extra servo/regulator trade-off parameter c in the expression for the integral time,

$$\tau_i = \min\{\tau, c(\tau_c + \theta)\},\tag{25}$$

where c = 4 gives the original SIMC rule. However, introducing an extra parameter adds complexity, and the potential performance benefit of approximately 10% (see Figure 7) does not seem sufficiently large to justify it.

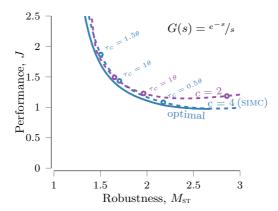


Figure 14: PI control: Evaluation of suggested modification c = 2 in (24) for integrating process.

Nevertheless, one may consider choosing another (lower) fixed value for c, and Haugen (2010) suggests using c=2 to improve performance for input disturbances. If we use the recommended tuning $\tau_c=\theta$, we find indeed that IAE performance J is improved compared to SIMC (see Figure 14). However, robustness is worse, with $M_{\rm ST}$ close to 2 (where the SIMC rule gives $M_{\rm ST}$ close to 1.7). More importantly, as seen from Figure 14 the SIMC performance is better if we decrease τ_c to get the same robustness in terms of $M_{\rm ST}$. In fact, SIMC is closer to the Pareto optimal curve for most values of $M_{\rm ST}$. Actually, a better fixed value would be c=3. However, changing the parameter c causes the recommended tuning $\tau_c=\theta$ to shift to the less robust region. In summary, we find that the value c=4 in the original SIMC rule provides a well balanced servo/regulator trade-off. To improve performance for input disturbances on an integrating process, we recommend decreasing the tuning constant τ_c/θ , say to around 0.7, rather than changing the value of c.

6.4 iSIMC for second-order plus delay process

For a second-order plus delay process,

$$G = \frac{ke^{-\theta s}}{(\tau_1 s + 1)(\tau_2 s + 1)} \tag{26}$$

where $\tau_1 > \tau_2$, the original SIMC rule gives a PID controller on the series form (2) with

$$k_c = \frac{1}{k} \frac{\tau}{(\tau_c + \theta)}, \quad \tau_i = \min\{\tau, 4(\tau_c + \theta)\}, \quad \tau_d = \tau_2.$$
 (27)

The direct extension of the isimc rule would be to add another derivative term, $(\frac{\theta}{3}s+1)$, to the numerator of the PID controller in (2). First, this would not be a standard industrial controller and, second, it would give even more aggressive input usage. Thus, to get a PID controller, the following modified derivative time is recommended *

isimc:
$$\tau_d = \tau_2 + \theta/3$$
 (28)

with the controller gain and integral time as given in (27). To get a good trade-off between performance and robustness, we may select $\tau_c = \theta$, but τ_c may be reduced towards $\theta/2$ for processes where τ_2 is smaller than $\theta/3$. Again, to get settings for the parallel (ideal) PID-controller in (3) one must compute the factor $f = 1 + \tau_d/\tau_i$, and use (4).

6.5 Ziegler-Nichols tuning rule

We also show in Figure 9 by red triangles the location of the classical Ziegler and Nichols (1942) (ZN) PI and PID controllers. The ZN-tunings are obtained by first bringing the process to sustained oscilattions using a P-controller and recording the resulting "ultimate" period and controller gain. Since the ZN rules have no tuning parameter we get a single point in Figure 9. With exception of the pure time delay process (where ZN-PID is unstable and ZN-PI has very poor performance), the IAE performance for ZN is very good, but the ZN controllers are located in the "flat" trade-off region with poor robustness (large $M_{\rm ST}$ value).

The Ziegler-Nichols PID tuning rules were the by far most used rules for about 50 years, up to about 1990. The very poor performance of the ZN rules for pure time delay processes may then partly explain the myth that "time delay compensators", such as the Smith Predictor, may give significant performance benefits compared to PI- or PID-control for processes with large time delays (Grimholt and Skogestad, 2018).

^{*} If τ_2 is very large, specifically if $\tau_2 > 4(\tau_c + \theta)$, then one should approximate the process as a double integrating process, $G(s) \approx k'' e^{-\theta s}/s^2$ with $k'' = k/(\tau_1 \tau_2)$, and use the PTD-tunings for a double integrating process (Skogestad, 2003).

7 Conclusion

The IAE-optimal PI- and PID-settings for a first-order plus delay process (1) are shown for various robustness levels (as expressed by the M_s -value) in Figures 4 and 5, respectively. However, in practice, we recommend using the SIMC-rules for PI- and PID-tuning.

For PI-control, Figure 7 shows that the "original" SIMC rule in (15) (Skogestad, 2003) gives close-to optimal PI-performance. That is, by adjusting the tuning constant τ_c to get a desired robustnes, we can closely track the Pareto-optimal trade-off curve between performance and robustness. The only exception is for delay-dominant FOPTD processes, where the SIMC proportional gain is too small, but this can be corrected for by using the iSIMC-PI rule in (17).

For PID-control, we propose the iSIMC rule where derivative action with $\tau_d = \theta/3$ (16) is added. Note that this is for the cascade PID-controller in (2). Figure 9 shows that this rule gives close-to optimal PID-performance, even for delay-dominant processes. For a pure time delay process, the iSIMC PID-controller is an ID-controller which can by rewritten to give the iSIMC-PI rule in (17).

The improved performance/robustness trade-off of the *i*SIMC-PI and *i*SIMC rules, comes at the expense of increased input usage in response to measurement noise, output disturbances and setpoint changes. Thus, for most industrial cases where output performance is not the main concern, the original SIMC rule may be the best choice.

Bibliography

S Alcántara, C Pedret, and R Vilanova. On the model matching approach to PID design: analytical perspective for robust servo/regulator tradeoff tuning. *Journal of Process Control*, 20(5):596–608, 2010.

Karl Johan Åström and Tore Hägglund. *Advanced PID control*. The Instrumentation, Systems, and Automation Society; Research Triangle Park, NC 27709, 2006.

K.J. Åström, C.C. Hang, P. Persson, and W.K. Ho. *Towards Intelligent PID Control*, volume 28. 1992.

- W. L. Bialkowski. Control of the pulp and paper making process. In William S. Levine, editor, *The Control Handbook*, chapter 72, pages 1219–1243. CRC Press and IEEE Press, 1996.
- Stephen P. Boyd and Craig H. Barratt. *Linear controller design: limits of performance*. Prentice Hall Englewood Cliffs, NJ, 1991.
- E. B. Dahlin. Designing and tuning digital controllers. *Instruments and Control systems*, 41(6):77–83, 1968.
- David Di Ruscio. On tuning PI controllers for integrating plus time delay systems. *Modeling, Identification and Control*, 31(4):145–164, 2010.
- Olof Garpinger and Tore Hägglund. A software tool for robust PID design. In *Proc. 17th IFAC World Congress, Seoul, Korea, 2008.*
- Olof Garpinger, Tore Hägglund, and Karl Johan Åström. Performance and robustness trade-offs in PID control. *Journal of Process Control*, 24(5):568–577, 2014.
- Chriss Grimholt and Sigurd Skogestad. Optimal PI-control and verification of the SIMC tuning rule. In *IFAC conference on Advances in PID control (PID'12)*. The International Federation of Automatic Control, March 2012a.
- Chriss Grimholt and Sigurd Skogestad. The SIMC method for smooth pid controller tuning. In Ramon Vilanova and Antonio Visioli, editors, *PID control in the third Millennium Lessons Learned and new approaches*. Springer, 2012b.
- Chriss Grimholt and Sigurd Skogestad. Optimal PID-control on first order plus time delay systems & verification of the simc rules. In *10th IFAC International Symposium on Dynamics and Control of Process Systems*, 2013.
- Chriss Grimholt and Sigurd Skogestad. Optimization of fixed order controllers using exact gradients. *Unpublished, submitted to Journal of Process Control*, 2015.
- Chriss Grimholt and Sigurd. Skogestad. Should we forget the smith predictor? In 3rd IFAC conference on Advances in PID control, Ghent, Belgium, 9-11 May 2018., 2018.

- Finn Haugen. Comparing PI tuning methods in a real benchmark temperature control system. *Modeling, Identification and Control*, 31(3):79–91, 2010.
- Mikulas Huba. Performance measures, performance limits and optimal PI control for the IPDT plant. *Journal of Process Control*, 23(4):500–515, 2013.
- Birgitta Kristiansson and Bengt Lennartson. Robust and optimal tuning of PI and PID controllers. *IEE Proceedings-Control Theory and Applications*, 149(1): 17–25, 2002.
- Birgitta Kristiansson and Bengt Lennartson. Evaluation and simple tuning of PID controllers with high-frequency robustness. *Journal of Process Control*, 16(2):91–102, 2006.
- Jietae Lee, Wonhui Cho, and Thomas F. Edgar. Simple analytic PID controller tuning rules revisited. *Industrial & Engineering Chemistry Research*, 53(13): 5038–5047, 2014.
- Aidan O'Dwyer. *Automatic Tuning of PID Controllers*. Instrument Society of America (ISA), 1988.
- Aidan O'Dwyer. *Handbook of PI and PID controller tuning rules*. Imperial College Press, 2 edition, 2006.
- Danlel E. Rivera, Manfred Morarl, and Sigurd Skogestad. Internal model control. 4. PID controller design. *Ind. Eng. Chem. Process Des. Dev.*, 25: 252–256, 1986.
- Vanessa Romero Segovia, Tore Hägglund, and Karl Johan Åström. Design of measurement noise filters for PID control. In *IFAC World Congress* 2014, volume 19, pages 8359–8364, 2014.
- Sigurd Skogestad. Simple analytic rules for model reduction and PID controller tuning. *Journal of process control*, 13(4):291–309, 2003.
- John G. Ziegler and Nathaniel B. Nichols. Optimum settings for automatic controllers. *Trans. ASME*, 64(11):759–768, 1942.