

Simple rules for PID tuning

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Summary

- Main message: Can usually do much better by taking a systematic approach
- Key: Look at initial part of step response
Initial slope: $k' = k / \tau_1$
- SIMC tuning rules (“Skogestad IMC”)(*)
One tuning rule! Easily memorized

$$K_c = \frac{1}{k'} \cdot \frac{1}{(\theta + \tau_c)}$$
$$\tau_I = \min(\tau_1, 4(\tau_c + \theta))$$

$\tau_c \geq 0$: desired closed-loop response time (tuning parameter)
For robustness select: $\tau_c \geq$

Reference: S. Skogestad, “Simple analytic rules for model reduction and PID controller design”, *J.Proc.Control*, Vol. 13, 291-309, 2003

(*) “Probably the best simple PID tuning rules in the world”

Need a model for tuning

- Model: Dynamic effect of change in input u (MV) on output y (CV)
- First-order + delay model for PI-control

$$G(s) = \frac{k}{\tau_1 s + 1} e^{-\theta s}$$

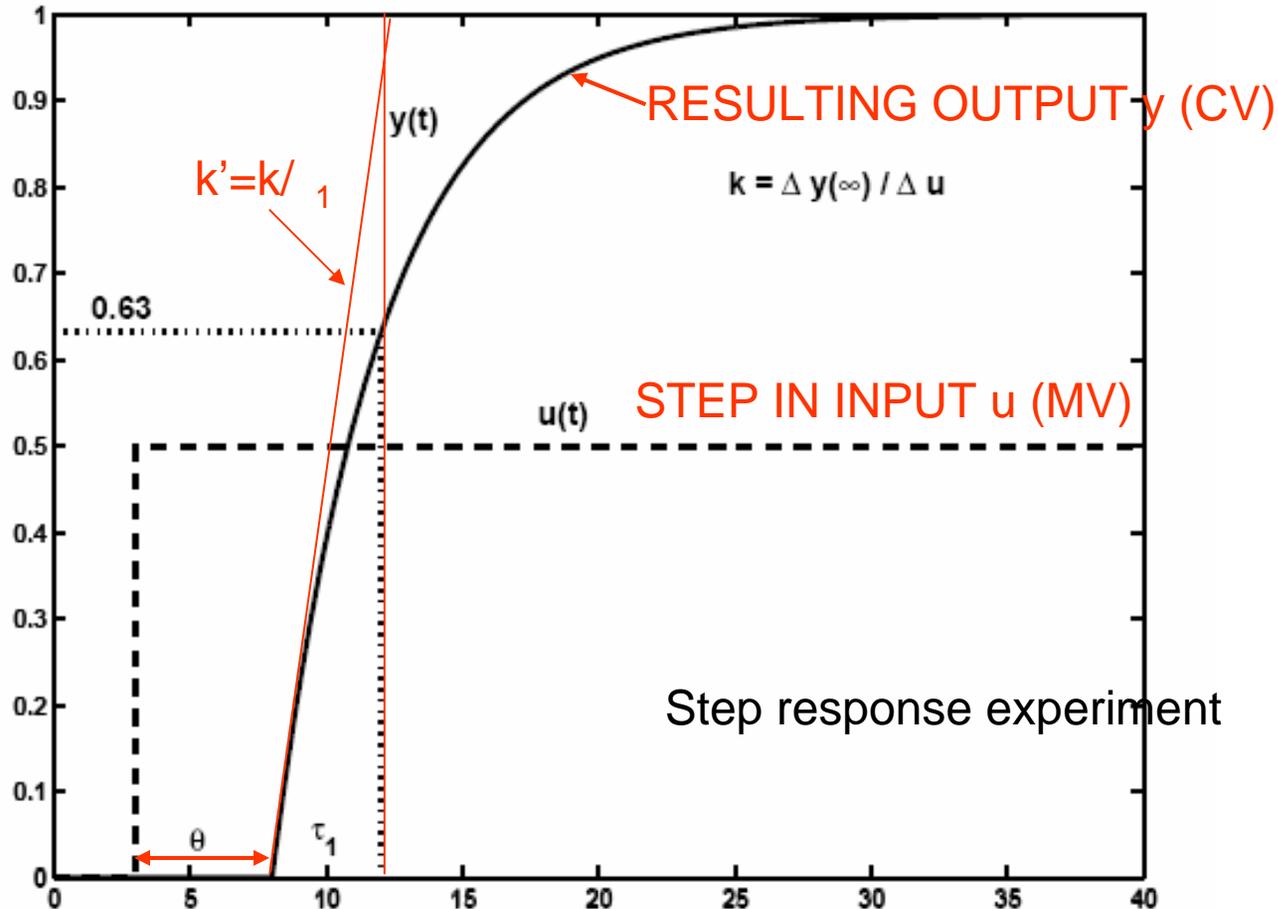
- Second-order model for PID-control

$$G(s) = \frac{k}{(\tau_1 s + 1)(\tau_2 s + 1)} e^{-\theta s}$$

Step response experiment

- Make step change in one u (MV) at a time
- Record the output (s) y (CV)

First-order plus delay process



Delay - Time where output does not change

τ_1 : Time constant - Additional time to reach 63% of final change

k : steady-state gain = $y(\infty) / u$

k' : slope after response "takes off" = k / τ_1

Model reduction of more complicated model

- Start with complicated stable model on the form

$$G_0(s) = k_0 \frac{(T_{10}s+1)(T_{20}s+1)\cdots}{(\tau_{10}s+1)(\tau_{20}s+1)\cdots} e^{-\theta_0 s}$$

- Want to get a simplified model on the form

$$G(s) = \frac{k}{(\tau_1 s+1)(\tau_2 s+1)} e^{-\theta s}$$

- Most important parameter is usually the “effective” delay

OBTAINING THE EFFECTIVE DELAY θ

Basis (Taylor approximation):

$$e^{-\theta s} \approx 1 - \theta s \quad \text{and} \quad e^{-\theta s} = \frac{1}{e^{\theta s}} \approx \frac{1}{1 + \theta s}$$

Effective delay =

“true” delay

+ inverse reponse time constant(s)

+ half of the largest neglected time constant (the “half rule”)
(this is to avoid being too conservative)

+ all smaller high-order time constants

The “other half” of the largest neglected time constant is added to τ_1
(or to τ_2 if use second-order model).

Example E1. *The process*

$$g_0(s) = \frac{1}{(s+1)(0.2s+1)}$$

is approximated as a first-order time delay process, $g(s) = ke^{-\theta s+1}/(\tau_1 s+1)$, with $k = 1$, $\theta = 0.2/2 = 0.1$ and $\tau_1 = 1 + 0.2/2 = 1.1$.

Example

$$g_0(s) = k \frac{(-0.3s + 1)(0.08s + 1)}{(2s + 1)(1s + 1)(0.4s + 1)(0.2s + 1)(0.05s + 1)^3}$$

half rule

is approximated as a first-order delay process with

$$\tau_1 = 2 + 1/2 = 2.5$$

$$\theta = 1/2 + 0.4 + 0.2 + 3 \cdot 0.05 + 0.3 - 0.08 = 1.47$$

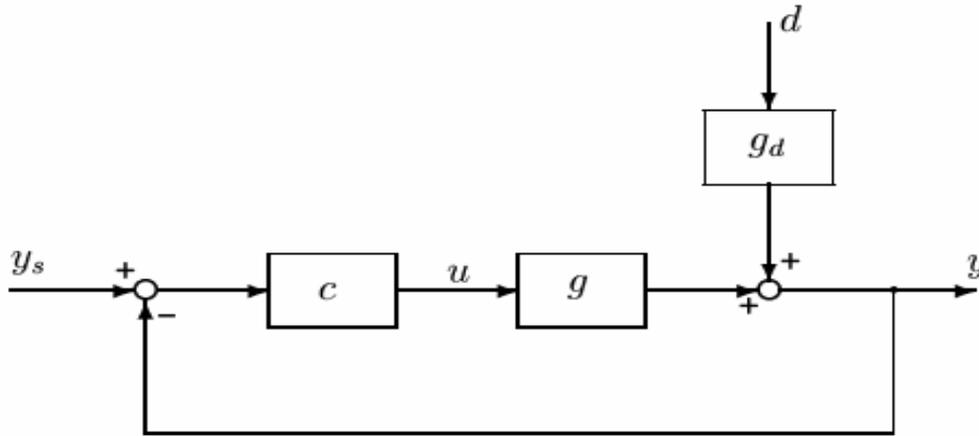
or as a second-order delay process with

$$\tau_1 = 2$$

$$\tau_2 = 1 + 0.4/2 = 1.2$$

$$\theta = 0.4/2 + 0.2 + 3 \cdot 0.05 + 0.3 - 0.08 = 0.77$$

Derivation of rules: Direct synthesis (IMC)



Closed-loop response to setpoint change

$$y = T y_s; \quad T(s) = \frac{gc}{1+gc}$$

Idea: Specify desired response $(y/y_s)=T$ and from this get the controller. Algebra:

$$c = \frac{1}{g} \cdot \frac{1}{\frac{1}{T}-1}$$

IMC Tuning = Direct Synthesis

- Controller: $c(s) = \frac{1}{g(s)} \cdot \frac{1}{\frac{1}{(y/y_s)_{\text{desired}}} - 1}$
- Consider second-order with delay plant: $g(s) = k \frac{e^{-\theta s}}{(\tau_1 s + 1)(\tau_2 s + 1)}$
- Desired first-order setpoint response: $\left(\frac{y}{y_s}\right)_{\text{desired}} = \frac{1}{\tau_c s + 1} e^{-\theta s}$
- Gives a “Smith Predictor” controller: $c(s) = \frac{(\tau_1 s + 1)(\tau_2 s + 1)}{k} \frac{1}{(\tau_c s + 1 - e^{-\theta s})}$
- To get a PID-controller use $e^{-\theta s} \approx 1 - \theta s$ and derive

$$c(s) = \frac{(\tau_1 s + 1)(\tau_2 s + 1)}{k} \frac{1}{(\tau_c + \theta)s}$$

which is a cascade form PID-controller with

$$K_c = \frac{1}{k} \frac{\tau_1}{\tau_c + \theta}; \quad \tau_I = \tau_1; \quad \tau_D = \tau_2$$

- τ_c is the sole tuning parameter

Integral time

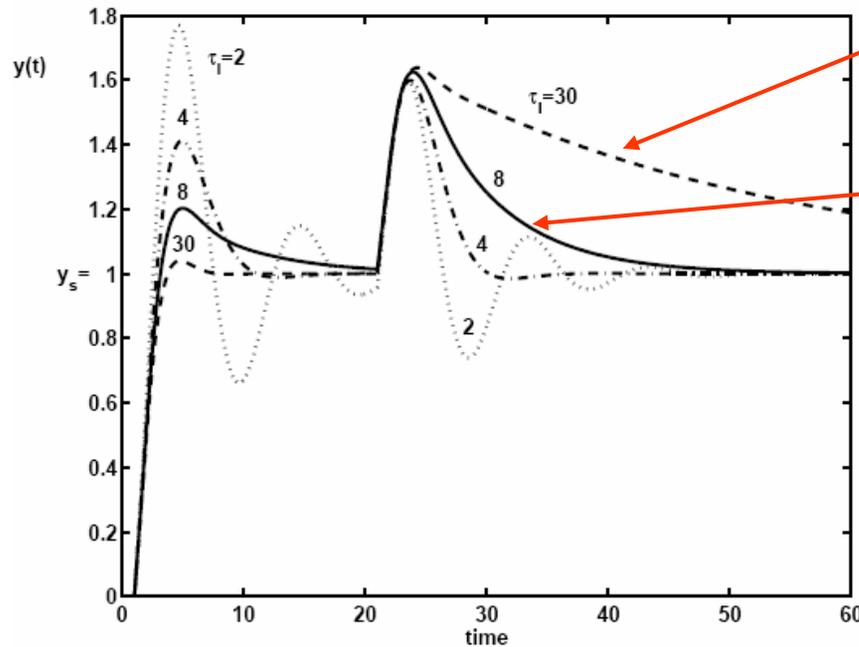
- Found:

Integral time = dominant time constant ($T_I = T_1$)

- Works well for setpoint changes

- Needs to be modify (reduce) T_I for “integrating disturbances”

Example: Integral time for “slow”/integrating process



IMC rule:
 $\tau_I = \tau_c = 30$

- Reduce τ_I to improve performance
- To just avoid slow oscillations:
 $\tau_I = 4(\tau_c + \tau_d) = 8$

(see derivation next page)

Figure 2: Effect of changing the integral time τ_I for PI-control of “slow” process $g(s) = e^{-s}/(30s + 1)$ with $K_c = 15$. Load disturbance of magnitude 10 occurs at $t = 20$.

Too large integral time: Poor disturbance rejection
 Too small integral time: Slow oscillations

Derivation integral time:

Avoiding slow oscillations for integrating process

- Integrating process: τ_I large
- Assume τ_I large and neglect delay
 - $G(s) = k e^{-s} / (\tau_I s + 1) \approx k / (\tau_I s) = k'/s$
- PI-control: $C(s) = K_c (1 + 1/\tau_I s)$
- Poles (and oscillations) are given by roots of closed-loop polynomial
 - $1+GC = 1 + k'/s \cdot K_c(1+1/\tau_I s) = 0$
 - or $\tau_I s^2 + k' K_c \tau_I s + k' K_c = 0$
 - Can be written on standard form $(s^2 + 2\zeta\omega_0 s + \omega_0^2)$ with
$$\tau_0 = \sqrt{\tau_I / (k' K_c)}; \zeta = \frac{1}{2} \sqrt{K_c \cdot k' \cdot \tau_I}$$
- To avoid oscillations must require $\zeta \geq 1$:
 - $K_c \cdot k' \cdot \tau_I \geq 4$ or $\tau_I \geq 4 / (K_c k')$
 - With choice $K_c = (1/k') (1/(\tau_c + \tau_I))$ this gives $\tau_I \geq 4 (\tau_c + \tau_I)$
- Conclusion integrating process: Want τ_I small to improve performance, but must be larger than $4 (\tau_c + \tau_I)$ to avoid slow oscillations

Summary: SIMC-PID Tuning Rules

For cascade form PID controller:

$$K_c = \frac{1}{k} \frac{\tau_1}{\tau_c + \theta} = \frac{1}{k'} \cdot \frac{1}{\tau_c + \theta} \quad (1)$$

$$\tau_I = \min\left\{\tau_1, \frac{4}{k' K_c}\right\} = \min\{\tau_1, 4(\tau_c + \theta)\} \quad (2)$$

$$\tau_D = \tau_2 \quad (3)$$

Derivation:

1. First-order setpoint response with response time τ_c (IMC-tuning = “Direct synthesis”)
2. Reduce integral time to get better disturbance rejection for slow or integrating process (but avoid slow cycling $\Rightarrow \tau_I \geq \frac{4}{k' K_c}$)

One tuning parameter: τ_c

Some special cases

Process	$g(s)$	K_c	τ_I	$\tau_D^{(4)}$
First-order	$k \frac{e^{-\theta a}}{(\tau_1 s + 1)}$	$\frac{1}{k} \frac{\tau_1}{\tau_c + \theta}$	$\min\{\tau_1, 4(\tau_c + \theta)\}$	-
Second-order, eq.(4)	$k \frac{e^{-\theta a}}{(\tau_1 s + 1)(\tau_2 s + 1)}$	$\frac{1}{k} \frac{\tau_1}{\tau_c + \theta}$	$\min\{\tau_1, 4(\tau_c + \theta)\}$	τ_2
Pure time delay ⁽¹⁾	$k e^{-\theta s}$	0	0 (*)	-
Integrating ⁽²⁾	$k' \frac{e^{-\theta s}}{s}$	$\frac{1}{k'} \cdot \frac{1}{(\tau_c + \theta)}$	$4(\tau_c + \theta)$	-
Integrating with lag	$k' \frac{e^{-\theta s}}{s(\tau_2 s + 1)}$	$\frac{1}{k'} \cdot \frac{1}{(\tau_c + \theta)}$	$4(\tau_c + \theta)$	τ_2
Double integrating ⁽³⁾	$k'' \frac{e^{-\theta s}}{s^2}$	$\frac{1}{k''} \cdot \frac{1}{4(\tau_c + \theta)^2}$	$4(\tau_c + \theta)$	$4(\tau_c + \theta)$

Table 1: SIMC PID-settings (23)-(25) for some special cases of (4) (with τ_c as a tuning parameter).

- (1) The pure time delay process is a special case of a first-order process with $\tau_1 = 0$.
- (2) The integrating process is a special case of a first-order process with $\tau_1 \rightarrow \infty$.
- (3) For the double integrating process, integral action has been added according to eq.(27).
- (4) The derivative time is for the series form PID controller in eq.(1).
- (*) Pure integral controller $c(s) = \frac{K_I}{s}$ with $K_I \stackrel{\text{def}}{=} \frac{K_c}{\tau_I} = \frac{1}{k(\tau_c + \theta)}$.

One tuning parameter: c

DERIVATIVE ACTION ?

First order with delay plant ($\tau_2 = 0$) with $\tau_c = \theta$:

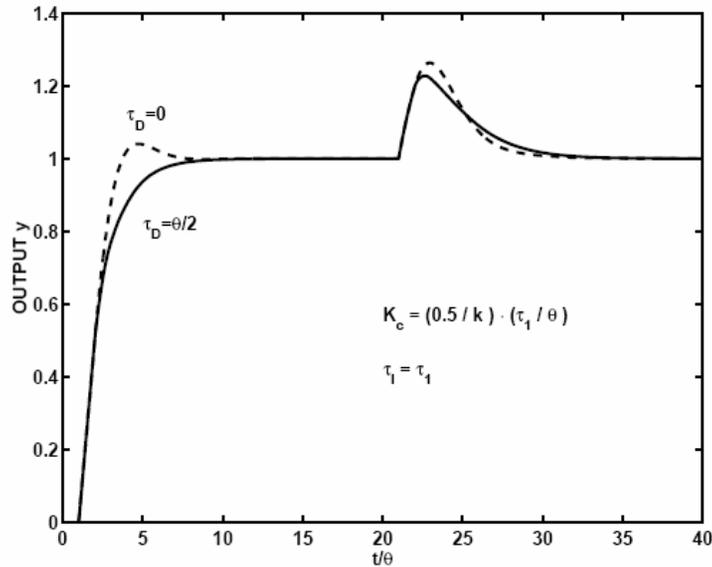


Figure 5: Setpoint change at $t = 0$. Load disturbance of magnitude 0.5 occurs at $t = 20$.

- Observe: Derivative action (solid line) has only a minor effect.
- Conclusion: Use second-order model (and derivative action) only when $\tau_2 > \theta$ (approximately)

Note: Derivative action is commonly used for temperature control loops. Select τ_D equal to time constant of temperature sensor

Selection of tuning parameter c

Two cases

1. **Tight control:** Want “fastest possible control”
subject to having good robustness
2. **Smooth control:** Want “slowest possible control”
subject to having acceptable disturbance rejection

TIGHT CONTROL

TUNING FOR FAST RESPONSE WITH GOOD ROBUSTNESS

$$\text{SIMC : } \tau_c = \theta \quad (4)$$

Gives:

$$K_c = \frac{0.5\tau_1}{k\theta} = \frac{0.5}{k'} \cdot \frac{1}{\theta} \quad (5)$$

$$\tau_I = \min\{\tau_1, 8\theta\} \quad (6)$$

$$\tau_D = \tau_2 \quad (7)$$

Try to memorize!

Gain margin about 3

Process $g(s)$	$\frac{k}{\tau_1 s + 1} e^{-\theta s}$	$\frac{k'}{s} e^{-\theta s}$
Controller gain, K_c	$\frac{0.5\tau_1}{k\theta}$	$\frac{0.5}{k'} \frac{1}{\theta}$
Integral time, τ_I	τ_1	8θ
Gain margin (GM)	3.14	2.96
Phase margin (PM)	61.4°	46.9°
Allowed time delay error, $\Delta\theta/\theta$	2.14	1.59
Sensitivity peak, M_s	1.59	1.70
Complementary sensitivity peak, M_t	1.00	1.30
Phase crossover frequency, $\omega_{180} \cdot \theta$	1.57	1.49
Gain crossover frequency, $\omega_c \cdot \theta$	0.50	0.51

Table 1: Robustness margins for first-order and integrating delay process using SIMC-tunings in (5) and (6) ($\tau_c = \theta$). The same margins apply to second-order processes if we choose $\tau_D = \tau_2$.

TIGHT CONTROL

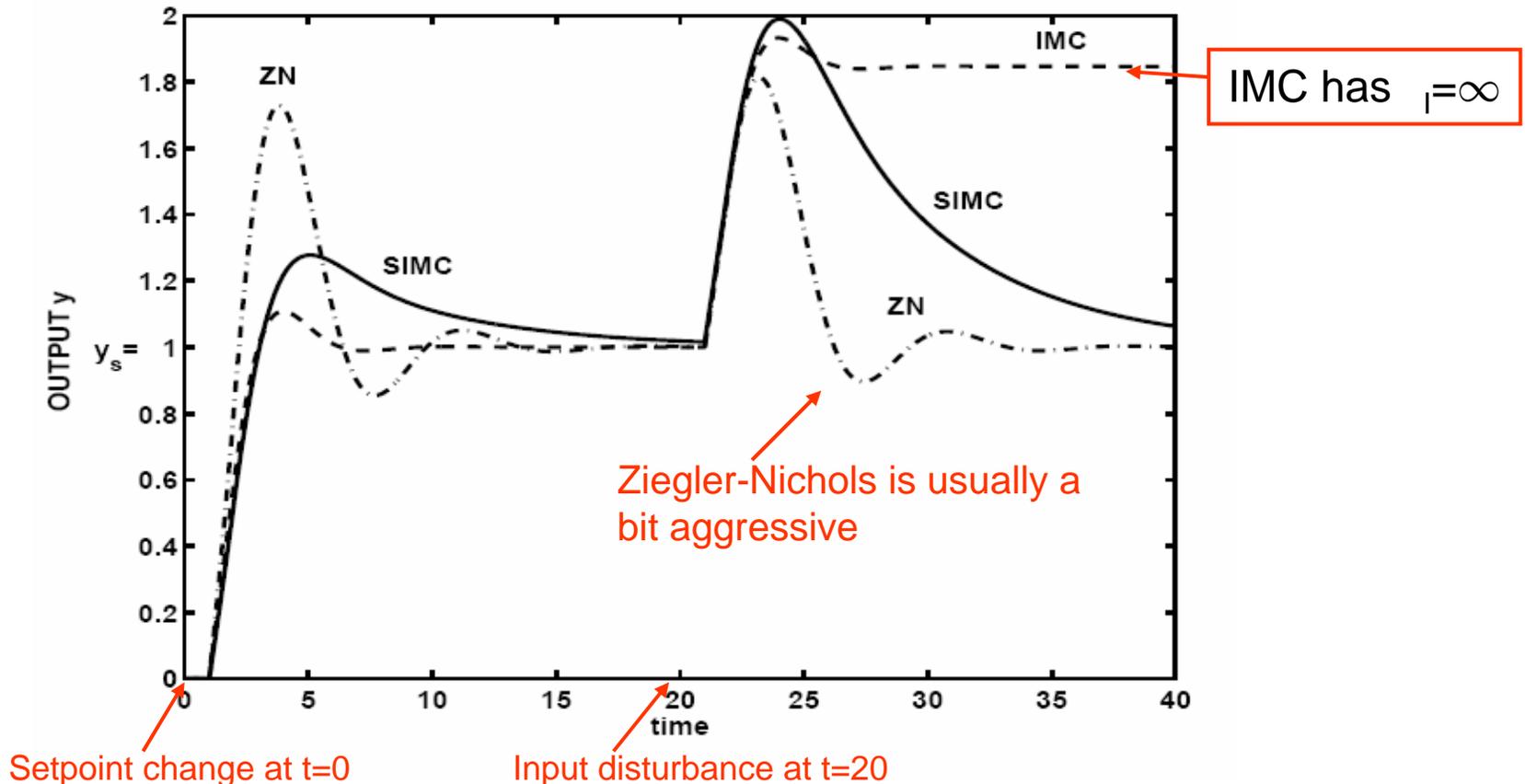
Example. Integrating process with delay=1. $G(s) = e^{-s}/s$.

Model: $k'=1$, $\tau_c=1$, $\tau_1=\infty$

SIMC-tunings with τ_c with $\tau_c=1$:

$$K_c = \frac{1}{k'} \frac{1}{\tau_c + \theta} = 1 \cdot \frac{1}{1+1} = 0.5$$

$$\tau_I = \min(\tau_1, 4(\tau_c + \theta)) = \min(\infty, 8) = 8$$



SMOOTH CONTROL

Minimum controller gain:

$$|c(j\omega)| \geq \frac{|u_0(j\omega)|}{|y_{\max}|}$$

Industrial practice: Variables (instrument ranges) often scaled such that

$$|u_0| \approx |y_{\max}| = \text{max range (span)}$$

Minimum controller gain is then

$$|c(j\omega)| \geq 1 \Rightarrow K_c \geq 1 \text{ for PID-control}$$

*Minimum gain for smooth control \Rightarrow
Common default factory setting $K_c=1$ is reasonable !*

Level control is often difficult...

- Typical story:
 - Level loop starts oscillating
 - Operator detunes by decreasing controller gain
 - Level loop oscillates even more
 -
- ???
- Explanation: Level is by itself unstable and requires control.

How avoid oscillating levels?

- Simplest: Use P-control only (no integral action)
- If you insist on integral action, then make sure the controller gain is sufficiently large
- If you have a level loop that is oscillating then use *Sigurds rule* (can be derived):

To avoid oscillations, increase $K_c \cdot \tau_i$ by factor

$$f = 0.1 \cdot (P_o / \tau_{i0})^2$$

where

P_o = period of oscillations [s]

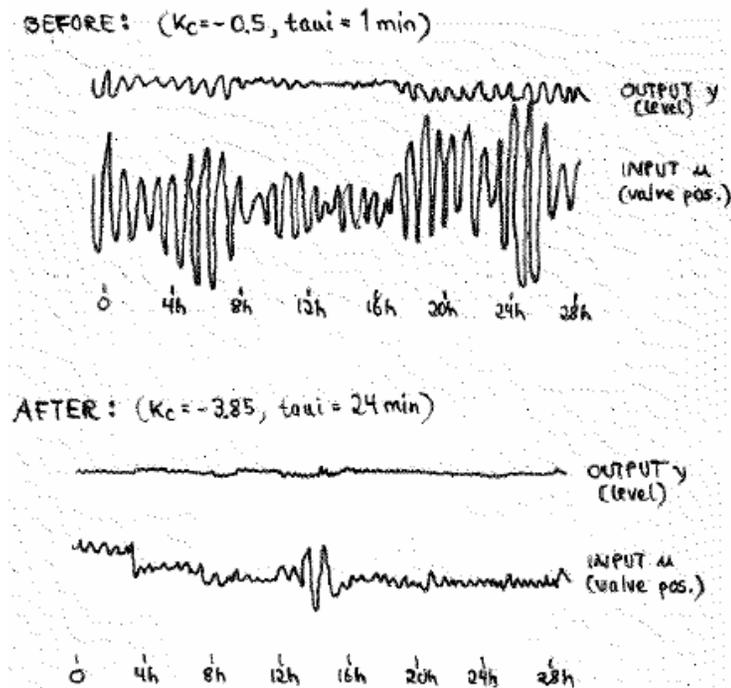
τ_{i0} = original integral time [s]

APPLICATION: RETUNING FOR INTEGRATING PROCESS

To avoid “slow” oscillations the product of the controller gain and integral time should be increased by factor $f \approx 0.1(P_0/\tau_{I0})^2$.

Real Plant data:

$$\text{Period of oscillations } P_0 = 0.85h = 51\text{min} \Rightarrow f = 0.1 \cdot (51/1)^2 = 260$$



Conclusion PID tuning

SIMC tuning rules

$$K_c = \frac{1}{k'} \cdot \frac{1}{(\theta + \tau_c)}$$
$$\tau_I = \min(\tau_1, 4(\tau_c + \theta))$$

1. **Tight control:** Select $\tau_c = \theta$ corresponding to

$$K_{c,\max} = \frac{0.5}{k'} \frac{1}{\theta}$$

2. **Smooth control.** Select $K_c \geq K_{c,\min} = \frac{|u_0|}{|y_{\max}|}$

Note: Having selected K_c (or τ_c), the integral time τ_I should be selected as given above

CONCLUSION

- It is simple (one single rule for all processes)
- It is excellent for teaching (analytical)
- It works very well for all of “our” processes